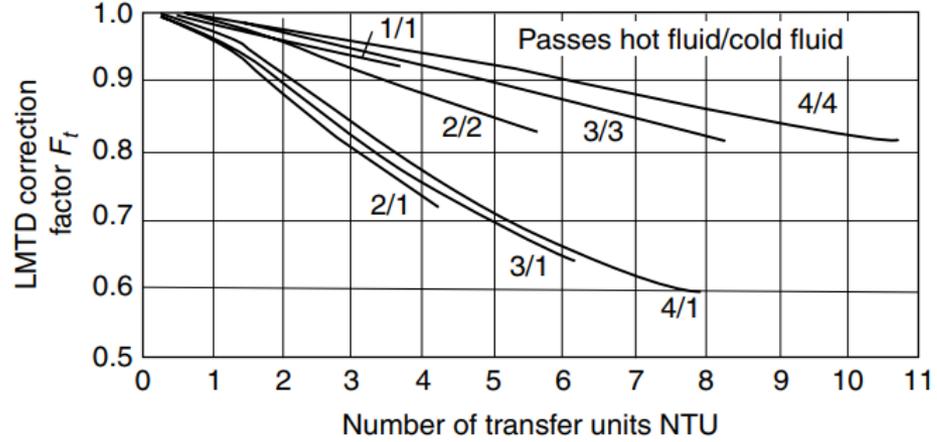


Intercambiadores de calor PHE. Consideraciones para el dimensionamiento: coeficientes pelliculares

$$G_p = \frac{\dot{m}_p}{Nchp \cdot b \cdot s} \quad D_H = \frac{4bs}{2(b+s)} \cong 2s; \quad b \gg s$$

$$Re = \frac{G_p D_H}{\mu} \quad Nu = \frac{h_p D_H}{\kappa_p} = C Re^a Pr^b \left(\frac{\mu}{\mu_{wall}} \right)^x$$

- | | |
|------------------------|-----------------------|
| En régimen turbulento | En régimen laminar |
| C: 0,15 a 0,40 (0,374) | C: 1,86 a 4,50 (3,18) |
| a: 0,65 a 0,85 (0,67) | a, b: 0,33 |
| b: 0,3 a 0,45 (0,33) | x: 0,05 a 0,20 (0,15) |
| x: 0,05 a 0,20 (0,15) | |

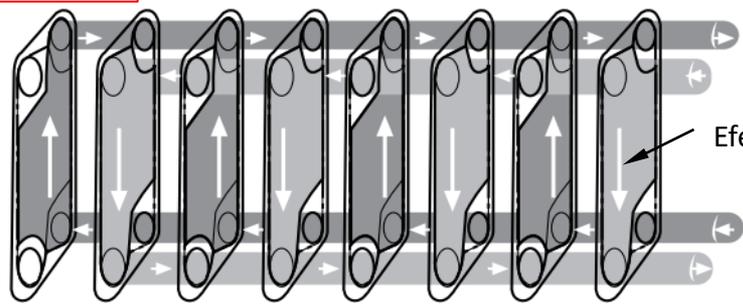


$$\dot{q} = U_d A_t F_t LMTD_{ctc}$$

$$U_d = \left[\frac{1}{h_H} + \frac{e_p}{\kappa_p} + \frac{1}{h_C} + r_{H,f} + r_{C,f} \right]^{-1}$$

$$A_t = \frac{\dot{q}}{U_d F_t LMTD_{ctc}}$$

$$NTU = \begin{cases} \frac{(T_{H,in} - T_{H,out})}{LMTD_{ctc}} & C_H = C_{min} \\ \frac{(T_{C,out} - T_{C,in})}{LMTD_{ctc}} & C_C = C_{min} \end{cases}$$



Efecto de extremo en 1-1

Intercambiadores de calor PHE. Consideraciones para el dimensionamiento: caída de presión (configuración 1 - 1)

$$L_t = \sqrt{L_p^2 + b^2}$$

$$G_p = \frac{\dot{m}_p}{Nchp \cdot b \cdot s}$$

$$\Delta P_p = f \left(\frac{L_t}{D_H} \right) \rho \frac{u_p^2}{2} = 8J_f \left(\frac{L_t}{D_H} \right) \rho \frac{G_p^2}{2\rho^2} = 8J_f \left(\frac{L_t}{D_H} \right) \frac{G_p^2}{2\rho}$$

$$\Delta P = \Delta P_p + \Delta P_{po}$$

$$J_f = 0,6Re^{-0,3}$$

$$Re = \frac{G_p D_H}{\mu}$$

$$D_H \cong 2s$$

$$\Delta P_{po} = 1,3 \frac{\rho u_h^2}{2} = 1,3 \frac{8\dot{m}_p^2}{\rho \pi^2 d_{hole}^4}$$

